

Hydraulic Pump Manufacture-Hydstar Hydraulic

液压泵专业生产厂家-江苏海斯特

Technical Manual 2023

Jiangsu Hydstar Hydraulic Technology Co., Ltd.

CONTENTS!

Piston Pump

Data sheet

Series 1 Size NG40 to 260 Nominal pressure 350 bar Maximum pressure 400 bar Open circuit

Contents

Features

– Variable axial piston pump of swashplate design for hydrostatic drives in open circuit hydraulic system.

– Designed primarily for use in mobile applications.

– The pump operates under self-priming conditions, with tank pressurization, or with an optional built-in charge pump (impeller).

- A comprehensive range of control options is available matching any application requirement.
- Power control option is externally adjustable, even when the pump is running.
- The through drive is suitable for adding gear pumps and axial piston pumps up to the same, i.e. 100% through drive.–
- The output flow is proportional to the drive speed and infinitely variable between $q_{V \text{ max}}$ and $q_{V \text{ min}} = 0$.

Ordering Code / Standard Program

In case of controls with several additional functions, observe the order of the columns, only one option per column is possible (e.g. LRDCH2). The following combinations are not available for the power control: LRDS2, LRDS5, L...GS, L...GS2, L...GS5, L...EC and the combination L...DG in conjunction with the stroke limiters H1, H2, H5, H6, U1 and U2.

Ordering Code / Standard Program

Ordering Code / Standard Program

¹⁾ S-shaft suitable for combination pump!

²⁾ To fit the flywheel case of the combustion engine

 $3)$ 2 \triangle 2-hole; 4 \triangle 4-hole

⁴⁾ Size 190 and 260 with $2 + 4$ -hole flange

 \bullet = available \circ = on request \circ = not available \Box = preferred program

Technical Data

Hydraulic fluid

Before starting project planning, please refer to our data sheets RE 90220 (mineral oil), RE 90221 (environmentally acceptable hydraulic fluids) and RE 90223 (HF hydraulic fluids) for detailed information regarding the choi-

ce of hydraulic fluid and operating conditions.

The variable pump A11VO is not suitable for operating with HFA, HFB and HFC. If HFD or environmentally acceptable hydraulic fluids are being used, the limitations regarding technical data and seals mentioned in RE 90221 and RE 90223 must be observed.

When ordering, please indicate the used hydraulic fluid.

Operating viscosity range

For optimum efficiency and service life, select an operating viscosity (at operating temperature) within the optimum range of

 v_{opt} = optimum operating viscosity 16 to 36 mm²/s

depending on the tank temperature (open circuit).

Limits of viscosity range

The limiting values for viscosity are as follows:

 $v_{\text{max}} = 1600 \text{ mm}^2/\text{s}$

Short-term $(t < 3$ min)

At cold start ($p \le 30$ bar, $n \le 1000$ rpm, $t_{min} = -40^{\circ}C$). Only for starting up without load. Optimum operating viscosity must be reached within approx. 15 minutes.

Note that the maximum hydraulic fluid temperature of 115°C must not be exceeded locally either (e.g. in the bearing area). The temperature in the bearing area is – depending on pressure and speed – up to 5 K higher than the average case drain temperature.

Special measures are necessary in the temperature range from -40°C and -25°C (cold start phase) , please contact us.

For detailed information about use at low temperatures, see RE 90300-03-B.

Selection diagram

Details regarding the choice of hydraulic fluid

The correct choice of hydraulic fluid requires knowledge of the operating temperature in relation to the ambient temperature: in an open circuit the tank temperature.

The hydraulic fluid should be chosen so that the operating viscosity in the operating temperature range is within the optimum range (v_{opt}) – see the shaded area of the selection diagram. We recommended that the higher viscosity class be selected in each case.

Example: At an ambient temperature of X°C an operating temperature of 60°C is set. In the optimum operating viscosity range (v_{opt}; shaded area) this corresponds to the viscosity classes VG 46 and VG 68; to be selected: VG 68.

Please note:

The case drain temperature, which is affected by pressure and speed, is always higher than the tank temperature. At no point in the system may the temperature be higher than 115°C.

If the above conditions cannot be maintained due to extreme operating parameters, please contact us.

Filtration

The finer the filtration, the higher the cleanliness level of the hydraulic fluid and the longer the service life of the axial piston unit.

To ensure functional reliability of the axial piston unit, the hydraulic fluid must have a claenliness level of at least

20/18/15 according to ISO 4406.

At very high hydraulic fluid temperatures (90°C to max. 115°C. not permitted for sizes 250 to 1000) at least cleanliness level

19/17/14 according to ISO 4406 is required.

If the above classes cannot be observed, please contact us.

Version *with* charge pump

Operating pressure range

Inlet

Absolute pressure at port S (suction port) Version *without* charge pump

Maximum permissible speed (speed limit)

Permissible speed by increasing the inlet pressure pabs at the suction port S or at $V_g \leq V_{g,max}$

Outlet

permissible for short-term $(t < 1s)$.

Minimum operating pressure

A minimum operating pressure $p_{B \min}$ is required in the pump service line depending on the speed, the swivel angle and the displacement (see diagram).

Case drain pressure

The case drain pressure at the ports T_1 and T_2 may be a maximum of 1.2 bar higher than the inlet pressure at the port S but not higher than

An unrestricted, full size case drain line directly to tank is required.

Temperature range of the shaft seal ring

The FKM shaft seal ring is permissible for case drain temperatures of -25° C to $+115^{\circ}$ C.

Note:

For applications below -25°C, an NBR shaft seal ring is necessary (permissible temperature range: -40°C to +90°C). State NBR shaft seal ring in clear text in the order.

Flushing the case

If a variable pump with control unit EP, HD, DR or stroke limiter (H., U.,) is operated over a long period ($t > 10$ min) with flow zero or operating pressure \leq 15 bar, flushing of the case via ports T_1 ", "T₂" or "R" is necessary.

Flushing the case is unnecessary in versions with charge pump (A11VLO), since a part of the charge flow is directed to the case.

Charge pump (impeller)

The charge pump is a circulating pump with which the A11VLO (size 130...260) is filled and therefore can be operated at higher speeds. This also simplifies cold starting at low temperatures and high viscosity of the hydraulic fluid. Tank charging is therefore unnecessary in most cases. A tank pressure of a max. 2 bar is permissible with charge pump.

Technical Data

Table of values (theoretical values, without efficiency and tolerances; values rounded)

¹⁾ The values apply at absolute pressure (p_{abs}) 1 bar at the suction port S and mineral hydraulic fluid.

²⁾ The values apply at absolute pressure (p_{abs}) of at least 0.8 bar at the suction port S and mineral hydraulic fluid.

³⁾ The values apply at $V_g \le V_{gmax}$ or in case of an increase in the inlet pressure p_{abs} at the suction port S (see diagram page 6)

 $4)$ – The area of validity is situated between 0 and the maximum permissible speed.

It applies for external stimuli (e.g. engine 2-8 times rotary frequency, cardan shaft twice the rotary frequency).

- The limit value applies for a single pump only.
- The loading on the connection parts has to be considered.

Caution:

Exceeding the permissible limit values could cause a loss of function, reduced service life or the destruction of the axial piston unit. The permissible values can be determined by calculation.

Technical Data

Permissible radial and axial loading on drive shaft

The values stated are maximum data and not permissible for continuous operation

Permissible input and through drive torques

1) Efficiency not considered

²⁾ For drive shafts with no radial force

³⁾ Observe max. input torque for shaft S!

Torque distribution

Determining the nominal value

Flow $q_v = \frac{V_g \cdot n \cdot \eta_v}{1000}$ I/min

Torque $T = \frac{V_g \cdot \Delta p}{20 \cdot \pi \cdot \eta_{\text{mh}}}$ Nm

Power
$$
P = \frac{2 \pi \cdot T \cdot n}{60,000} = \frac{q_v \cdot \Delta p}{600 \cdot n_t} \text{ kW}
$$

 V_g = Displacement per revolution in cm³

 $\Delta p =$ Differential pressure in bar

- n = Speed in rpm
- η_v = Volumetric efficiency

 η_{mh} = Mechanical-hydraulic efficiency

 η_t = Overall efficiency ($n_t = n_v \cdot n_{mh}$)

The power control regulates the displacement of the pump depending on the operating pressure so that a given drive power is not exceeded at constant drive speed.

 p_B • V_q = constant p_B = operating pressure V_a = displacement

The precise control with a hyperbolic control characteristic, provides an optimum utilization of available power.

The operating pressure acts on a rocker via a measuring piston. An externally adjustable spring force counteracts this, it determines the power setting.

If the operating pressure exceeds the set spring force, the control valve is actuated by the rocker, the pump swivels back (direction $V_{q min}$). The lever length at the rocker is shortened and the operating pressure can increase at the same rate as the displacement decreases without the drive powers being exceeded ($p_B \cdot V_g$ = constant).

The hydraulic output power (characteristic LR) is influenced by the efficiency of the pump.

State in clear text in the order:

- drive power P in kW
- drive speed n in rpm
- max. flow $q_{V \text{ max}}$ in l/min

After clarifying the details a power diagram can be created by our computer.

Characteristic LR

Circuit diagram LR

LRC Override with cross sensing

Cross sensing control is a summation power control system, whereby the total power, of both the A11VO and of a same size A11VO power controlled pump mounted onto the through drive, are kept constant.

If a pump is operating at pressures below the start of the control curve setting, then the surplus power not required, in a critical case up to 100%, becomes available to the other pump. Total power is thus divided between two systems as demand requires.

Any power being limited by means of pressure cut-off or other override functions is not taken into account.

Half side cross sensing function

When using the LRC control on the 1st pump (A11VO) and a power-controlled pump without cross sensing attached to the through drive, the power required for the 2nd pump is deducted from the setting of the 1st pump. The 2nd pump has priority in the total power setting.

The size and start of control of the power control of the 2nd pump must be specified for rating the control of the 1st pump.

Circuit diagram LRC

Size 40 ... 145

LR3 High-pressure related override

The high-pressure related power override is a total power control in which the power control setting is piloted by the load pressure of an attached fixed pump (port Z).

As a result the A11VO can be set to 100% of the total drive power. The power setting of the A11VO is reduced proportional to the load-dependent rise in operating pressure of the fixed pump. The fixed pump has priority in the total power setting.

The measuring area of the power reduction pilot piston is designed as a function of the size of the fixed pump.

Circuit diagram LR3

Size 40 ... 145

10/64

Size 190 ... 260

LG1/2 Pilot-pressure related override

This power control works by overriding the control setting with an external pilot pressure signal. This pilot pressure acts on the adjustment spring of the power regulator via port Z.

The mechanically adjusted basic setting can be hydraulically adjusted by means of different pilot pressure settings, enabling different power mode settings.

If the pilot pressure signal is then adjusted by means of an external power limiting control, the total hydraulic power consumption of all users can be adapted to the available drive power from the engine.

The pilot pressure used for power control is generated by an external control element that is not a component part of the A11VO (e.g. see also data sheet RE 95310, Electronic Load Limiting Control, LLC).

LG1 Negative power override

Power control with negative override, LG1: the force resulting from the pilot pressure is acting against the mechanical adjustment spring of the power control.

Increasing the pilot pressure reduces the power setting.

Circuit diagram LG1

Size 40 ... 145

Size 190 ... 260

LG2 Positive power override

Power control with positive override, LG2: the force resulting from the pilot pressure is additive the mechanical adjustment spring of the power control.

An increase in pilot pressure increases the power output.

Circuit diagram LG2

Size 40 ... 145

LE1/2 Electric override (negative)

Contrary to hydraulic power control override, the basic power setting is reduced by an electric pilot current applied to a proportional solenoid. The resulting force is acting against the mechanical power control adjustment spring.

The mechanically adjusted basic power setting can be varied by means of different control current settings.

 $Increase$ in current $=$ decrease in power

If the pilot current signal is adjusted by a load limiting control the power consumption of all actuators will be reduced to match the available power from the diesel engine.

A 12V (LE1) or 24V (LE2) supply is required for the control of the proportion solenoid.

Technical data - Solenoids

Circuit diagram LE1/2

Size 190 ... 260

Overview of power overrides

Effect of power overrides at rising pressure or current

LRD Power control with pressure cut-off

The pressure cut-off corresponds to a pressure control which adjusts the pump displacement back to $V_{\text{q min}}$, when the pressure setting is reached.

This function overrides the power control, i.e. below the preset pressure value, the power function is effective.

The pressure cut-off function is integrated into the pump control module and is preset to a specified value at the factory.

Setting range from 50 to 350 bar

Characteristic LRD

Circuit diagram LRD

Size 40...145

Size 190...260

LRE Power control with pressure cut-off, 2-stage

By connecting an external pilot pressure to port Y, the basic value of the pressure cut-off can be increased by 50 +20 bar and a 2nd pressure setting implemented.

This value is usually above the primary pressure relief valve setting and therefore disables the pressure cut-off function. The pressure signal at port Y must be between 20 and 50 bar.

Characteristic LRE

Circuit diagram LRE *Size 40...145*

Size 190...260

LRG Power control with pressure cut-off, hydraulically remote controlled

See page 21 for description and characteristic (pressure control remote controlled, DRG)

LRDS Power control with pressure cut-off and load sensing

The load sensing control is a flow control option that operates as a function of the load pressure to regulate the pump displacement to match the actuator flow requirement.

The flow depends here on the cross section of the external sensing orifice (1) fitted between the pump outlet and the actuator. The flow is independent of the load pressure below the power curve and the pressure cut-off setting and within the control range of the pump.

The sensing orifice is usually a separately arranged load sensing directional valve (control block). The position of the directional valve piston determines the opening cross section of the sensing orifice and thus the flow of the pump.

The load sensing control compares pressure before and after the sensing orifice and maintains the pressure drop across the orifice (differential pressure Δp) and with it the pump flow constant.

If the differential pressure Δp increases at the sensing orifice, the pump is swivelled back (towards $V_{q min}$), and, if the differential pressure Δp decreases, the pump is swivelled out (towards $V_{\text{g,max}}$) until the pressure drop across the sensing orifice in the valve is restored.

 $\Delta p_{\text{orifice}} = p_{\text{pump}} - p_{\text{actuator}}$

The setting range for Δp is between 14 bar and 25 bar.

The standard differential pressure setting is 18 bar. (Please state in clear text when ordering).

The stand-by pressure in zero stroke operation (sensing orifice plugged) is slightly above the Δp setting.

In a standard LS system the pressure cut-off is integrated in the pump control. In a LUDV (flow sharing) system the pressure cut-off is integrated in the LUDV control block.

(1) The sensing orifice (control block) is not included in the pump supply.

Characteristic LRDS

Circuit diagram LRDS

Size 190 ... 260

X $\left[\ast\right]$ (1) T_1 AM Τ **WALL** V_q max V_q min $\left| \cdot \right|$ ٦M $\overline{\mathsf{T}_2}$ M_1 S

LRS2 Power control with load sensing, electric override

This control option adds a proportional solenoid to override to the mechanically set load sensing pressure. The pressure differential change is proportional to the solenoid current.

Increasing current = smaller Δp -setting

See following characteristic for details (example). Please consult us during the project planning phase. For solenoid specification, see page 12 (LE2)

Characteristic LRS2

Circuit diagram LRS2

Size 40 ... 145

Size 190 ... 260

LRS5 Power control with load sensing, hydraulic override

This control option adds an external proportional pilot pressure signal (to port Z) to override the mechanically set load sensing pressure.

Increasing pilot pressure = smaller Δp-setting

See following characteristic for details (example). Please consult us during the project planning phase.

Characteristic LRS5

Circuit diagram LRS5

Size 40 ... 145

Size 190 ... 260

LR... Power control with stroke limiter

The stroke limiter can be used to vary or limit the displacement of the pump continuously over the whole control range. The displacement is set in LRH with the pilot pressure p_{St} (max. 40 bar) applied to port Y or in LRU by the control current applied to the proportional solenoid. A DC current of 12V (U1) or 24V (U2) is required to control the proportional solenoid.

The power control overrides the stoke limiter control, i.e. below the hyperbolic power characteristic, the displacement is controlled by the control current or pilot pressure. When exceeding the power characteristic with a set flow or load pressure, the power control overrides and reduces the displacement following the hyperbolic characteristic.

To permit operation of the pump displacement control from its starting position $V_{\text{q max}}$ to $V_{\text{q min}}$, a minimum control pressure of 30 bar is required for the electric stroke limiter LRU1/2 and the hydraulic stroke limiter LRH2/6.

The required control pressure is taken either from the load pressure, or from the externally applied control pressure at the G port.

To ensure functioning of the stroke limiter even at low operating pressure, port G must be supplied with external control pressure of approx. 30 bar.

Note:

If no external control pressure is connected at G, the shuttle valve must be removed.

Note

The spring return feature in the controller is not a safety device

The spool valve inside the controller can get stuck in an undefined position by internal contamination (contaminated hydraulic fluid, abrasion or residual contamination from system components). As a result, the axial piston unit can no longer supply the flow specified by the operator.

Check whether your application requires that remedial measures be taken on your machine in order to bring the driven consumer into a safe position (e.g. immediate stop).

LRH1/5 Hydraulic stroke limiter (negative characteristic)

HYDSTAR

With increasing pilot pressure the pump swivels to a smaller displacement.

Start of control (at $V_{\text{g max}}$), can be set ________ from 4 - 10 bar

State start of control in clear text in the order.

Starting position without control signal (pilot pressure): $V_{g \text{ max}}$

Characteristic H1

Increase in pilot pressure (Vg max – Vg min) ________Δp = 25 bar

Characteristic H5

Increase in pilot pressure $(V_{g \text{ max}} - V_{g \text{ min}})$ _________ $\Delta p = 10$ bar

Circuit diagram LRH1/5

Size 40 ... 145

Size 190 ... 260

Control from $V_{g \text{ max}}$ to $V_{g \text{ min}}$

LRH2/6 Hydraulic stroke limiter (positive characteristic)

Control from $V_{q min}$ to $V_{q max}$

With increasing pilot pressure the pump swivels to a higher displacement.

Start of control (at $V_{g min}$), can be set _________ from 4-10 bar

State start of control in clear text in the order.

Starting position without control signal (pilot pressure):

- at operating pressure and external control pressure $<$ 30 bar: $V_{g \text{ max}}$
- at operating pressure or external control pressure $>$ 30 bar: $V_{\alpha \text{ min}}$

Characteristic H2

Characteristic H6

$$
Increase in pilot pressure (V_{g min} - V_{g max}) \underline{\hspace{2cm}} \Delta p = 10 bar
$$

Circuit diagram LRH2/6

Size 190 ... 260

LRU1/2 Electric stroke limiter (positive characteristic)

Control from $V_{g \text{ min}}$ to $V_{g \text{ max}}$

With increasing control current the pump swivels to a higher displacement.

Technical data - solenoids

Starting position without control signal (control current):

 – at operating pressure and external control pressure $<$ 30 bar: $V_{q max}$

 – at operating pressure or external control pressure $>$ 30 bar: $V_{g \min}$

The following electronic controllers and amplifiers are available for actuating the proportional solenoids

– BODAS controller RC

– Analog amplifier RA ______________________ RE 95230

Characteristic LRU1/2

Circuit diagram LRU1/2

Size 40 ... 145

DR Pressure control

The pressure control keeps the pressure in a hydraulic system constant within its control range even under varying flow conditions. The variable pump only moves as much hydraulic fluid as is required by the actuators. If the operating pressure exceeds the setpoint set at the integral pressure control valve, the pump displacement is automatically swivelled back until the pressure deviation is corrected.

Starting position in depressurized state: $V_{g \text{ max}}$

Setting range from 50 to 350 bar.

Characteristic: DR

Circuit diagram DR

Size 190 ... 260

DRS Pressure control with load sensing

The load sensing control is a flow control option that operates as a function of the load pressure to regulate the pump displacement to match the actuator flow requirement.

The flow depends here on the cross section of the external sensing orifice (1) fitted between the pump outlet and the actuator. The flow is independent of the load pressure below the pressure cut-off setting and within the control range of the pump.

The sensing orifice is usually a separately arranged load sensing directional valve (control block). The position of the directional valve piston determines the opening cross section of the sensing orifice and thus the flow of the pump.

The load sensing control compares pressure before and after the sensing orifice and maintains the pressure drop across the orifice (differential pressure Δp) and with it the pump flow constant.

If the differential pressure Δp increases at the sensing orifice. the pump is swivelled back (towards $V_{q min}$), and, if the differential pressure Δp decreases, the pump is swivelled out (towards $V_{\text{q max}}$) until the pressure drop across the sensing orifice in the valve is restored.

 $\Delta p_{\text{orifice}} = p_{\text{pump}} - p_{\text{actuator}}$

The setting range for Δp is between 14 bar and 25 bar.

The standard differential pressure setting is 18 bar. (Please state in clear text when ordering).

The stand-by pressure in zero stroke operation (sensing orifice plugged) is slightly above the Δp setting.

(1) The sensing orifice (control block) is not included in the pump supply.

Characteristic: DRS

Circuit diagram DRS

DRG Pressure control, remote controlled

The remote control pressure cut-off regulator permits the adjustment of the pressure setting by a remotely installed pressure relief valve (1). Pilot flow for this valve is provide by a fixed orifice in the control module.

Setting range from 50 to 350 bar.

In addition the pump can be unloaded into a standby pressure condition by an externally installed 2/2-way directional valve (2).

Both functions can be used individually or in combination (see circuit diagram).

The external valves are not included in the pump supply.

As a separate pressure relief valve (1) we recommend:

DBDH 6 (manual control), see RE 25402

Characteristic: DRG

Note: The remote controlled pressure cut-off is also possible in combination with LR, HD and EP.

Circuit diagram DRG

Size 40 ... 145

Size 190 ... 260

DRL Pressure control for parallel operation

The pressure control DRL is suitable for pressure control of several axial piston pumps A11VO in parallel operation pumping into a common pressure header.

The parallel pressure control has a pressure rise characteristic of approx. 15 bar from $q_{v \, max}$ to $q_{v \, min}$. The pump regulates therefore to a pressure dependent swive angle. This results in stable control behavior, without the need of "staging" the individual pump compensators.

With the externally installed pressure relief valve (1) the nominal pressure setting of all pumps connected to the system is adjusted to the same value.

Setting range from 50 to 350 bar.

Each pump can be individually unloaded from the system by a separately installed 3/2-way directional valve (2).

The check valves (3) in the service line (port A) or control line (port X) must be provided generally.

The external valves are not included in the pump supply.

As a separate pressure relief valve (1) we recommend:

DBDH 6 (manual control), see RE 25402

Characteristic DRL

Circuit diagram DRL

Size 40 ... 145

HD – Hydraulic Control, Pilot-Pressure Related

With the pilot-pressure related control the pump displacement is adjusted in proportion to the pilot pressure applied to port Y. Maximum permissible pilot pressure $p_{St, max} = 40$ bar

Control from $V_{g min}$ to $V_{g max}$.

With increasing pilot pressure the pump swivels to a higher displacement.

Start of control (at $V_{\text{q min}}$), can be set _________ from 4-10 bar

State start of control in clear text in the order.

Starting position without control signal (pilot pressure):

- at operating pressure and external control pressure $<$ 30 bar: $V_{\text{g max}}$
- at operating pressure or external control pressure > 30 bar: $V_{g min}$

A control pressure of 30 bar is required to swivel the pump from its starting position $V_{\text{q max}}$ to $V_{\text{q min}}$.

The required control pressure is taken either from the load pressure, or from the externally applied control pressure at the G port.

To ensure the control even at low operating pressure < 30 bar the port G must be supplied with an external control pressure of approx. 30 bar.

Note:

If no external control pressure is connected at G, the shuttle valve must be removed.

Note

The spring return feature in the controller is not a safety device

The spool valve inside the controller can get stuck in an undefined position by internal contamination (contaminated hydraulic fluid, abrasion or residual contamination from system components). As a result, the axial piston unit can no longer supply the flow specified by the operator.

Check whether your application requires that remedial measures be taken on your machine in order to bring the driven consumer into a safe position (e.g. immediate stop).

Characteristic HD1

Increase in pilot pressure $V_{g min}$ to $V_{g max}$ __________ $\Delta p = 10$ bar

Characteristic HD2

Increase in pilot pressure $V_{g min}$ to $V_{g max}$ $\Delta p = 25$ bar

Circuit diagram HD

Size 40 ... 260

HD – Hydraulic Control, Pilot-Pressure Related

HD.D Hydraulic control with pressure cut-off

Circuit diagram HD.D

Size 40 ... 145

The pressure cut-off corresponds to a pressure control which adjusts the pump displacement back to $V_{g \text{ min}}$ when the pressure setting is reached.

This function overrides the HD control, i.e. the pilot-pressure related displacement control is functional below the pressure setting.

The pressure cut-off function is integrated into the pump control module and is preset to a specified value at the factory.

Setting range from 50 to 350 bar.

Pressure cut-off characteristic D

EP – Electric Control with Proportional Solenoid

With the electric control with proportional solenoid, the pump displacement is adjusted proportionally to the solenoid current, resulting in a magnetic control force, acting directly onto the control spool that pilots the pump control piston.

Control from $V_{q min}$ to $V_{q max}$

With increasing control current the pump swivels to a higher displacement.

Starting position wthout control signal (control current):

- at operating pressure and external control pressure $<$ 30 bar: $V_{\text{g max}}$
- at operating pressure or external control pressure > 30 bar: $V_{\text{c min}}$

A control pressure of 30 bar is required to swivel the pump from its starting position $V_{\text{g max}}$ to $V_{\text{g min}}$.

The required control pressure is taken either from the load pressure, or from the externally applied control pressure at port G.

To ensure the control even at low operating pressure < 30 bar the port G must be supplied with an external control pressure of approx. 30 bar.

Note:

If no external control pressure is connected at G, the shuttle valve must be removed.

Note:

Install pump with EP control in the oil tank only when using mineral hydraulic oils and an oil temperature in the tank of max. 80°C.

The following electronic control units and amplifiers are available for actuating the proportional solenoids

– BODAS controller RC

– Analog amplifier RA

RE 95230

Technical data, solenoid at EP1, EP2

Note

The spring return feature in the controller is not a safety device

The spool valve inside the controller can get stuck in an undefined position by internal contamination (contaminated hydraulic fluid, abrasion or residual contamination from system components). As a result, the axial piston unit can no longer supply the flow specified by the operator.

Check whether your application requires that remedial measures be taken on your machine in order to bring the driven consumer into a safe position (e.g. immediate stop).

Characteristic EP1/2

Circuit diagram EP1/2

Size 40 ... 260

EP – Electric Control with Proportional Solenoid

EP.D Electric control with pressure cut-off

The pressure cut-off corresponds to a pressure control which adjusts the pump displacement back to $V_{\text{q min}}$ when the pressure setting is reached.

This function overrides the EP control, i.e. the control current related displacement control is functional below the pressure setting.

The valve for the pressure cut-off is integrated in the control case and is set to a fixed specified pressure value at the factory.

Setting range from 50 to 350 bar

Pressure cut-off characteristic D

Circuit diagram EP.D

Size 40 ... 145

Size 190 ... 260

Dimensions, Size 40

Before finalizing your design, please request a certified drawing. Dimensions in mm.

LRDCS

Power control LR with pressure cut-off D, cross sensing control C and load sensing control S

Dimensions, Size 40

Before finalizing your design, please request a certified drawing. Dimensions in mm.

Shaft ends

Ports

¹⁾ Center bore according to DIN 332 (thread acc. to DIN 13)

²⁾ For max. tightening torque, please refer to general notes on page 64

3) ANSI B92.1a-1976, 30° pressure angle, flat root, side fit, tolerance class 5

4) Depending on adjustment data and operating pressure

⁵⁾ Depending on installation position, T1 or T2 must be connected (see also page 61)

O = Open, must be connected (closed on delivery)

 $X = Closed$ (in normal operation)

Dimensions, Size 40

LRDH1/LRDH5

Power control with pressure cut-off and hydraulic stroke limiter (negative characteristic)

LRDU1/LRDU2

Power control with pressure cut-off and electric stroke limiter (positive characteristic)

LG1E

Power control with pilot-pressure related override (negative) and 2-stage pressure cut-off

Before finalizing your design, please request a certified drawing. Dimensions in mm.

LRDH2/LRDH6

Power control with pressure cut-off and hydraulic stroke limiter (positive characteristic)

LR3DS

Power control with high-pressure related override, pressure cut-off and load sensing control

LG2E

Power control with pilot-pressure related override (positive) and 2-stage pressure cut-off

Dimensions, Size 40

HD1D/HD2D

Hydraulic control, pilot-pressure related with pressure cut-off

DRS/DRG

Pressure control with load sensing control Pressure control remote controlled

LE1S/LE2S

Power control with electric override (negative) and load sensing control

Before finalizing your design, please request a certified drawing. Dimensions in mm.

EP1D/EP2D

Electric control with proportional solenoid and pressure cut-off

DRL

Pressure control for parallel operation

LE2S2/LE1S5/LE2S5

Power control with electric override (negative) and load sensing control, override

Dimensions, Size 60

Before finalizing your design, please request a certified drawing. Dimensions in mm.

LRDCS

Power control LR with pressure cut-off D, cross sensing control C and load sensing control S

Dimensions, Size 60

Before finalizing your design, please request a certified drawing. Dimensions in mm.

Shaft ends

Ports

¹⁾ Center bore according to DIN 332 (thread acc. to DIN 13)

²⁾ For max. tightening torque, please refer to general notes on page 64

3) ANSI B92.1a-1976, 30° pressure angle, flat root, side fit, tolerance class 5

4) Depending on adjustment data and operating pressure

5) Depending on installation position, T1 or T2 must be connected (see also page 61)

O = Open, must be connected (closed on delivery)

 $X = Closed$ (in normal operation)

Dimensions, Size 60

LRDH1/LRDH5

Power control with pressure cut-off and hydraulic stroke limiter (negative characteristic)

LRDU1/LRDU2

Power control with pressure cut-off and electric stroke limiter (positive characteristic)

LG1E

Power control with pilot-pressure related override (negative) and 2-stage pressure cut-off

Before finalizing your design, please request a
certified drawing. Dimensions in mm.

LRDH2/LRDH6

Power control with pressure cut-off and hydraulic stroke limiter (positive characteristic)

LR3DS

Power control with high-pressure related override, pressure cut-off and load sensing control

LG2E

Power control with pilot-pressure related override (positive) and 2-stage pressure cut-off

Dimensions, Size 60

HD1D/HD2D

Hydraulic control, pilot-pressure related with pressure cut-off

DRS/DRG

Pressure control with load sensing control Pressure control remote controlled

LE1S/LE2S

Power control with electric override (negative) and load sensing control

Before finalizing your design, please request a certified drawing. Dimensions in mm.

EP1D/EP2D

Electric control with proportional solenoid and pressure cut-off

DRL

Pressure control for parallel operation

LE2S2/LE1S5/LE2S5

Power control with electric override (negative) and load sensing control, override

Dimensions, Size 75

Before finalizing your design, please request a certified drawing. Dimensions in mm.

LRDCS

Power control LR with pressure cut-off D, cross sensing control C and load sensing control S

Dimensions, Size 75

Before finalizing your design, please request a certified drawing. Dimensions in mm.

Shaft ends

Ports

1) Center bore according to DIN 332 (thread acc. to DIN 13)

²⁾ For max. tightening torque, please refer to general notes on page 64

3) ANSI B92.1a-1976, 30° pressure angle, flat root, side fit, tolerance class 5

4) Depending on adjustment data and operating pressure

5) Depending on installation position, T1 or T2 must be connected (see also page 61)

O = Open, must be connected (closed on delivery)

 $X = Closed$ (in normal operation)

Dimensions, Size 75

LRDH1/LRDH5

Power control with pressure cut-off and hydraulic stroke limiter (negative characteristic)

LRDU1/LRDU2

Power control with pressure cut-off and electric stroke limiter (positive characteristic)

LG1E

Power control with pilot-pressure related override (negative) and 2-stage pressure cut-off

Before finalizing your design, please request a
certified drawing. Dimensions in mm.

LRDH2/LRDH6

Power control with pressure cut-off and hydraulic stroke limiter (positive characteristic)

LR3DS

Power control with high-pressure related override, pressure cut-off and load sensing control

LG2E

Power control with pilot-pressure related override (positive) and 2-stage pressure cut-off

Dimensions, Size 75

HD1D/HD2D

Hydraulic control, pilot-pressure related with pressure cut-off

DRS/DRG

Pressure control with load sensing control Pressure control remote controlled

LE1S/LE2S

Power control with electric override (negative) and load sensing control

Before finalizing your design, please request a certified drawing. Dimensions in mm.

EP1D/EP2D

Electric control with proportional solenoid and pressure cut-off

DRL

Pressure control for parallel operation

LE2S2/LE1S5/LE2S5

Power control with electric override (negative) and load sensing control, override

Dimensions, Size 95

Before finalizing your design, please request a certified drawing. Dimensions in mm.

LRDCS

Power control LR with pressure cut-off D, cross sensing control C and load sensing control S

Dimensions, Size 95

Before finalizing your design, please request a certified drawing. Dimensions in mm.

Shaft ends

75

ø50

55

R2.5

Ports

ø101

1) Center bore according to DIN 332 (thread acc. to DIN 13)

²⁾ For max. tightening torque, please refer to general notes on page 64

3) ANSI B92.1a-1976, 30° pressure angle, flat root, side fit, tolerance class 5

4) Depending on adjustment data and operating pressure

5) Depending on installation position, T1 or T2 must be connected (see also page 61)

O = Open, must be connected (closed on delivery)

 $X = Closed$ (in normal operation)

Dimensions, Size 95

LRDH1/LRDH5

Power control with pressure cut-off and hydraulic stroke limiter (negative characteristic)

LRDU1/LRDU2

Power control with pressure cut-off and electric stroke limiter (positive characteristic)

LG1E

Power control with pilot-pressure related override (negative) and 2-stage pressure cut-off

Before finalizing your design, please request a certified drawing. Dimensions in mm.

LRDH2/LRDH6

Power control with pressure cut-off and hydraulic stroke limiter (positive characteristic)

LR3DS

Power control with high-pressure related override, pressure cut-off and load sensing control 342.5

LG2E

Power control with pilot-pressure related override (positive) and 2-stage pressure cut-off

324

Dimensions, Size 95

HD1D/HD2D

Hydraulic control, pilot-pressure related with pressure cut-off

DRS/DRG

Pressure control with load sensing control Pressure control remote controlled

LE1S/LE2S

Power control with electric override (negative) and load sensing control

Before finalizing your design, please request a certified drawing. Dimensions in mm.

EP1D/EP2D

Electric control with proportional solenoid and pressure cut-off

DRL

Pressure control for parallel operation

LE2S2/LE1S5/LE2S5

Power control with electric override (negative) and load sensing control, override

Dimensions, Size 130/145

Before finalizing your design, please request a certified drawing. Dimensions in mm.

LRDCS

Power control LR with pressure cut-off D, cross sensing control C and load sensing control S

²⁾ The case or length dimension with flange SAE 3 is 5 mm shorter than the standard case.

Dimensions, Size 130/145

Before finalizing your design, please request a certified drawing. Dimensions in mm.

Shaft ends

75

55

Ports

¹⁾ Center bore according to DIN 332 (thread acc. to DIN 13)

2) For max. tightening torque, please refer to general notes on page 64

3) ANSI B92.1a-1976, 30° pressure angle, flat root, side fit, tolerance class 5

4) Depending on adjustment data and operating pressure

5) Depending on installation position, T1 or T2 must be connected (see also page 61)

6) with charge pump

 $X = Closed$ (in normal operation)

327

O = Open, must be connected (closed on delivery)

Dimensions, Size 130/145

LRDH1/LRDH5

Power control with pressure cut-off and hydraulic stroke limiter (negative characteristic)

LRDU1/LRDU2

Power control with pressure cut-off and electric stroke limiter (positive characteristic)

LG1E

Power control with pilot-pressure related override (negative) and 2-stage pressure cut-off

Before finalizing your design, please request a certified drawing. Dimensions in mm.

LRDH2/LRDH6

Power control with pressure cut-off and hydraulic stroke limiter (positive characteristic)

LR3DS

Power control with high-pressure related override, pressure cut-off and load sensing control

LG2E

Power control with pilot-pressure related override (positive) and 2-stage pressure cut-off

Dimensions, Size 130/145

HD1D/HD2D

Hydraulic control, pilot-pressure related with pressure cut-off

DRS/DRG

Pressure control with load sensing control Pressure control remote controlled

LE1S/LE2S

Power control with electric override (negative) and load sensing control

Before finalizing your design, please request a certified drawing. Dimensions in mm.

EP1D/EP2D

Electric control with proportional solenoid and pressure cut-off

DRL

Pressure control for parallel operation

LE2S2/LE1S5/LE2S5

Power control with electric override (negative) and load sensing control, override

Dimensions, Size 190

Before finalizing your design, please request a certified drawing. Dimensions in mm.

LRDCS

Power control LR with pressure cut-off D, cross sensing control C and load sensing control S

Dimensions, Size 190

Before finalizing your design, please request a certified drawing. Dimensions in mm.

Shaft ends

Ports

¹⁾ Center bore according to DIN 332 (thread acc. to DIN 13)

 $^{2)}$ For max. tightening torque, please refer to general notes on page 64

³⁾ ANSI B92.1a-1976, 30° pressure angle, flat root, side fit, tolerance class 5

4) Depending on adjustment data and operating pressure

5) Depending on installation position, T1 or T2 must be connected (see also page 61)

6) with charge pump

O = Open, must be connected (closed on delivery)

 $X = Closed$ (in normal operation)

Dimensions, Size 190

LRDH1/LRDH5

Power control with pressure cut-off and hydraulic stroke limiter (negative characteristic)

LRDU1/LRDU2

Power control with pressure cut-off and electric stroke limiter (positive characteristic)

LG1EH

Power control with pilot-pressure related override (neg.), 2-stage pressure cut-off and hydr. stroke limiter

LRDH2/LRDH6

Power control with pressure cut-off and hydraulic stroke limiter (positive characteristic)

Before finalizing your design, please request a certified drawing. Dimensions in mm.

LR3DS

Power control with high-pressure related override, pressure cut-off and load sensing control

LG2EH

Power control with pilot-pressure related override (pos.), 2-stage pressure cut-off and hydr. stroke limiter

Before finalizing your design, please request a certified drawing. Dimensions in mm.

Dimensions, Size 190

HD1D/HD2D

Hydraulic control, pilot-pressure related with pressure cut-off

DRS/DRG

Pressure control with load sensing control Pressure control remote controlled

LE1S/LE2S

Power control with electric override (negative) and load sensing control

EP1D/EP2D

Electric control with proportional solenoid and pressure cut-off

DRL

Pressure control for parallel operation

LE2S2/LE1S5/LE2S5

Power control with electric override (negative) and load sensing control, override

333

Dimensions, Size 260

Before finalizing your design, please request a certified drawing. Dimensions in mm.

LRDCS

Power control LR with pressure cut-off D, cross sensing control C and load sensing control S

334

Dimensions, Size 260

Before finalizing your design, please request a certified drawing. Dimensions in mm.

Shaft ends

Ports

¹⁾ Center bore according to DIN 332 (thread acc. to DIN 13)

²⁾ For max. tightening torque, please refer to general notes on page 64

3) ANSI B92.1a-1976, 30° pressure angle, flat root, side fit, tolerance class 5

4) Depending on adjustment data and operating pressure

⁵⁾ Depending on installation position, T1 or T2 must be connected (see also page 61)

6) with charge pump

 $X = Closed$ (in normal operation)

O = Open, must be connected (closed on delivery)

Dimensions, Size 260 Before finalizing your design, please request a
certified drawing. Dimensions in mm.

LRDH1/LRDH5

Power control with pressure cut-off and hydraulic stroke limiter (negative characteristic)

LRDU1/LRDU2

Power control with pressure cut-off and electric stroke limiter (positive characteristic)

LG1EH

Power control with pilot-pressure related override (neg.), 2-stage pressure cut-off and hydr. stroke limiter

certified drawing. Dimensions in mm.

LRDH2/LRDH6

Power control with pressure cut-off and hydraulic stroke limiter (positive characteristic)

LR3DS

Power control with high-pressure related override, pressure cut-off and load sensing control

LG2EH

Power control with pilot-pressure related override (pos.), 2-stage pressure cut-off and hydr. stroke limiter

Before finalizing your design, please request a certified drawing. Dimensions in mm.

Dimensions, Size 260

HD1D/HD2D

Hydraulic control, pilot-pressure related with pressure cut-off

DRS/DRG

Pressure control with load sensing control Pressure control remote controlled

LE1S/LE2S

Power control with electric override (negative) and load sensing control

EP1D/EP2D

Electric control with proportional solenoid and pressure cut-off

DRL

Pressure control for parallel operation

LE2S2/LE1S5/LE2S5

Power control with electric override (negative) and load sensing control, override

337

Through Drive Dimensions

3/4 in 11T 16/32 DP¹⁾ (SAE J744 - 19-4 (A-B)) K52

ersion with charge pump

Flange SAE J744 – 82-2 (A) Coupler for splined shaft acc. to ANSI B92.1a-1976 5/8 in 9T 16/32 DP¹⁾ (SAE J744 – 16-4 (A) K01

Flange SAE J744 – 101-2 (B) Coupler for splined shaft acc. to ANSI B92.1a-1976 7/8 in 13T 16/32 DP1) (SAE J744 – 22-4 (B)) K02 1 in 15T 16/32 DP1) (SAE J744 – 25-4 (B-B))K04

In size 190 and 260 the hole template is turned 45° counter-clockwise.

*) Version with charge pump

Flange SAE J744 – 127-2 (C) Coupler for splined shaft acc. to ANSI B92.1a-1976 1 1/4 in 14T 12/24 DP¹⁾ (SAE J744 – 32-4 (C)) K07 1 1/2 in 17T 12/24 DP¹⁾ (SAE J744 – 38-4 (C-C)) **K24**
W30x2x30x14x9a **K80** Coupler for splined shaft acc. to DIN 5480 W30x2x30x14x9g

 $A₂$ O-Ring2) A3 ø127 € 181 213 A1 up to mounting flange

*) Version with charge pump

Note:

The mounting flange may be turned through 90°. Standard position as illustrated. Please state in clear text if required.

¹⁾ 30° pressure angle, flat root, side fit, tolerance class 5

2) O-ring included in the delivery contents

3) DIN 13, for max. tightening torque, please refer to general notes on page 64

Through Drive Dimensions

Flange SAE J744–127-2+4 (A) Coupler for splined shaft acc. to ANSI B92.1a-19761 1/4 in 14T 12/24 DP1) (SAE J744 – 32-4 (C) K07 1 1/2 in 17T 12/24 DP1) (SAE J744 – 38-4 (C-C)) K24

*) Version with charge pump

Flange SAE J744 – 101-2 (E) Coupler for splined shaft acc. to ANSI B92.1a-1976 1 3/4 in 13T 16/32 DP1) (SAE J744 – 32-4 (C)) K72 Coupler for splined shaft acc. to DIN 5480 W50x2x30x24x9g) K84

*) Version with charge pump

Note:

The mounting flange may be turned through 90°. Standard position as illustrated. Please state in clear text if required.

- $1)$ 30° pressure angle, flat root, side fit, tolerance class 5
- 2) O-ring included in the delivery contents
- ³⁾ DIN 13, for max. tightening torque, please refer to general notes on page 64

Overview of Attachments for A11V(L)O

¹⁾ We recommends special versions of the gear pumps. Please ask.

²⁾ Only A10VO with 4-hole mounting flange can be mounted to A11V(L)O 190 and 260.

Combination Pumps A11VO + A11VO

Total length $A¹$)

¹⁾ When using the Z shaft (splined shaft DIN 5480) for the attached pump (2nd pump)

2) Version with charge pump

When ordering combination pumps, the type designations of the 1st and 2nd pumps must be connected by a "+". Ordering code 1st pump + Ordering code 2nd pump

Ordering example:

A11VO130LRDS/10R-NZD12K61 + A11VO60LRDS/10R-NZC12N00

Swivel Angle Indicator

Optical swivel angle indicator, V

With the optical swivel angle indicator, a mechanical pointer on the side of the pump case displays the position of the swivel angle of the pump.

Electric swivel angle sensor, R

With the electric swivel angle indicator the swivel position of the pump is measured by an electric swivel angle sensor. It has a robust, sealed case and integrated electronics designed for automotive applications.

As an output the Hall effect swivel angle sensor supplies a voltage signal proportional to the swivel angle (see technical parameters).

Mating connector

AMP Superseal 1.5; 3-pin,

The mating connector is not included in the delivery contents.

Connector for Solenoids

DEUTSCH DT04-2P-EP04, 2-pin

molded, without bidirectional suppressor diode (standard)

Type of protection according to DIN/EN 60529: IP67 and IP69K

Circuit diagram symbol

without bidirectional suppressor diode

Mating connector

DEUTSCH DT06-2S-EP04

– 2 female connectors ___________________ 0462-201-16141

The mating connector is not included in the delivery contents.

Note for round solenoids:

The position of the connector can be changed by turning the solenoid body.

Proceed as follows:

- 1. Loosen fixing nut (1)
- 2. Turn the solenoid body (2) to the desired position.
- 3. Tighten the fixing nut Tightening torque of fixing nut: 5+1 Nm (width across the flats WAF 26, 12kt DIN 3124)

Installation Notes

General

During commissioning and operation, the axial piston unit must be filled with hydraulic fluid and air bled. This is also to be observed following a relatively long standstill as the system may empty via the hydraulic lines.

The case drain in the case interior must be directed to the tank via the highest tank port (T_1, T_2) . The minimum suction pressure at port S must not fall below 0.8 bar absolute (without charge pump) or 0.6 bar (with charge pump).

In all operational conditions, the suction line and case drain line must flow into the tank below the minimum fluid level.

Installation position

See examples below. Additional installation positions are available upon request.

Below-tank installation (standard)

Pump below the minimum fluid level of the tank. Recommended installation positions: 1 and 2.

Above-tank installation

Pump above the minimum fluid level of the tank.

Observe the maximum permissible suction height $h_{s max} = 800$ mm.

The version A11VLO (with charge pump) is not designed for installation above the tank.

Recommendation for installation position 7 (shaft up): A check valve in the case drain line (opening pressure 0.5 bar) can prevent the case interior from draining.

For control options with pressure control, displacement limiters, HD and EP control, the minimum displacement setting must be $V_q \geq 5\% V_{q \text{ max}}$.

 $h_{\text{s max}} = 800$ mm, $h_{\text{t min}} = 200$ mm, $h_{\text{min}} = 100$ mm, $SB =$ Silencer plate (baffle plate)

When designing the tank, ensure adequate space a_{min} between the suction line and the case drain line to prevent the heated, returned fluid from being directly drawn back out.

Installation Notes

Tank installation

Pump below the minimum fluid level in the tank.

 $h_{\rm s max}$ = 800 mm, $h_{\rm t min}$ = 200 mm, $h_{\rm min}$ = 100 mm, SB = Silencer plate (baffle plate)

When designing the tank, ensure adequate space a_{min} between the suction line and the case drain line to prevent the heated, returned fluid from being directly drawn back out.

A11VO Series

Notice

General Notes

- The A11VO pump is designed to be used in open circuits.
- Project planning, assembly and commissioning of the axial piston unit require the involvement of qualified personnel.
- The service line ports and function ports are only designed to accommodate hydraulic lines.
- During and shortly after operation, there is a risk of burns on the axial piston unit and especially on the solenoids. Take appropriate safety measures (e. g. by wearing protective clothing).
- Depending on the operational state of the axial piston unit (operating pressure, fluid temperature), the characteristic may shift.
- Pressure ports:

The ports and fixing threads are designed for the specified maximum pressure. The machine or system manufacturer must ensure that the connecting elements and lines correspond to the specified operating conditions (pressure, flow, hydraulic fluid, temperature) with the necessary safety factors.

- The data and notes contained herein must be adhered to.
- The following tightening torques apply:
- Threaded hole for axial piston unit: The maximum permissible tightening torques $M_{G \text{ max}}$ are maximum values for the threaded holes and must not be exceeded. For values, see the following table.
- Fittings: -

Observe the manufacturer's instruction regarding the tightening torques of the used fittings.

- Fixing screws:

For fixing screws according to DIN 13, we recommend checking the tightening torque individually according to VDI 2230.

- Locking screws:

For the metal locking screws supplied with the axial piston unit, the required tightening torques of locking screws M_V apply. For values, see the following table.

– The product is not approved as a component for the safety concept of a general machine according to DIN EN ISO 13849.

